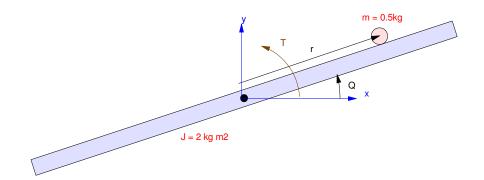
ECE 463/663 - Homework #7

Servo Compensators. Due Monday, March 8th



The dynamics of a Ball and Beam System (homework set #4) with a disturbance are

$$s\begin{bmatrix} r\\ \theta\\ \dot{r}\\ \dot{\theta}\end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0\\ 0 & 0 & 0 & 1\\ 0 & -7 & 0 & 0\\ -1.96 & 0 & 0 & 0\end{bmatrix}\begin{bmatrix} r\\ \theta\\ \dot{r}\\ \dot{\theta}\end{bmatrix} + \begin{bmatrix} 0\\ 0\\ 0\\ 0.4\end{bmatrix}T + \begin{bmatrix} 0\\ 0\\ 0\\ 0.4\end{bmatrix}d$$

Full-State Feedback with Constant Disturbances

1) For the nonlinear simulation, use the feedback control law you computed in homework #6

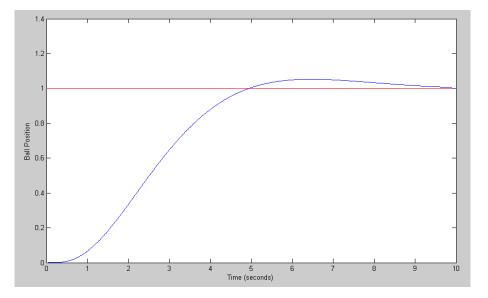
- With R = 1 and the mass of the ball = 0.5kg (same result you got for homework #6), and
- With R = 1 and the mass of the ball increased to 0.6kg

(i.e. a constant disturbance on the system due to the extra mass of the ball)

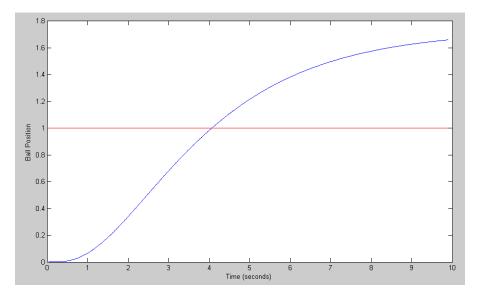
From homework #6

Kx = -7.2211 48.8540 -5.6397 20.0000 Kr = -2.3211

Plugging these in to the nonlinar simulation with m = 0.5kg and 0.6kg

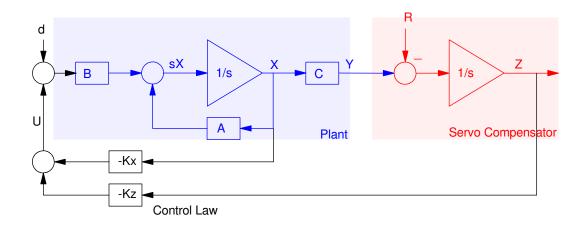


m = 0.5kg step response (nonlinear simulation)



m = 0.6kg step response (nonlinear simulation)

Servo Compensators with Constant Set-Points



2) Assume a constant disturbance and/or a constant set point. Design a feedback control law that results in

- The ability to track a constant set point (R = constant)
- The ability to reject a constant disturbance (d = constant),
- A 2% settling time of 6 seconds, and
- No overshoot for a step input.

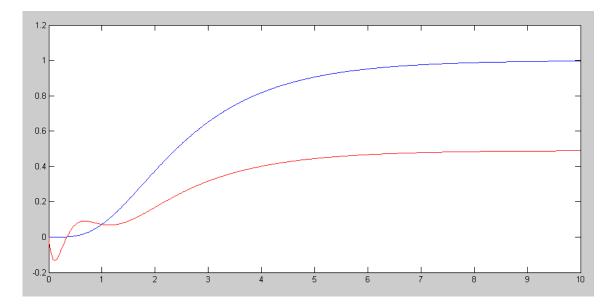
In matlab

```
>> A = [0, 0, 1, 0; 0, 0, 0, 1; 0, -7, 0, 0; -1.96, 0, 0, 0];
>> B = [0; 0; 0; 0.4];
>> C = [1, 0, 0, 0];
>> A5 = [A, 0*B; C, 0]
          0
                           1.0000
                     0
                                                       0
                                            0
                                      1.0000
          0
                                                       0
                     0
                                 0
               -7.0000
          0
                                 0
                                            0
                                                       0
   -1.9600
                                 0
                                            0
                                                       0
                     0
    1.0000
                     0
                                 0
                                                       0
                                            0
>> B5 = [B;0]
          0
          0
          0
    0.4000
          0
>> B5r = [0*B; -1]
     0
     0
     0
     0
    -1
>> K5 = ppl(A5, B5u, [-0.67,-2, -3, -4, -5])
                     Кx
                                                      Kz
K5 = -84.6071 200.9500 -71.9893
                                         36.6750
                                                   -28.7143
```

3) For the linear system, plot the step response

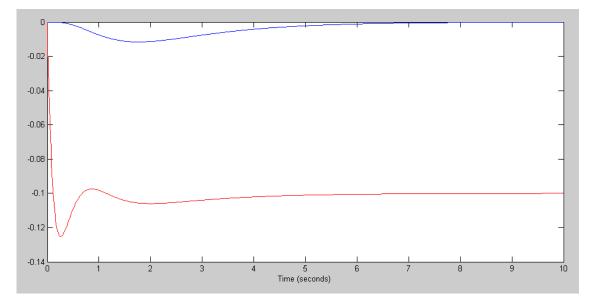
Step response with respect to R: (Just for fun, plot both position and input (U))

```
>> C5 = [C,0 ; -K5];
>> D5 = [0;0];
>> G5 = ss(A5 - B5*K5, B5r, C5, D5);
>> y = step(G5,t);
>> plot(t,y(:,1),'b',t,y(:,2)/10,'r');
```



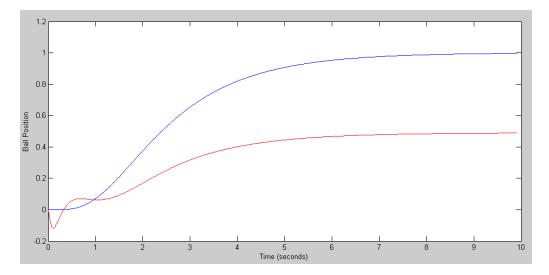
Step Resposne with respect to R: Position (blue) & U/10 (red) The system can track a constant set point

```
>> G5 = ss(A5 - B5*K5, B5, C5, D5);
>> y = step(G5,t);
>> plot(t,y(:,1),'b',t,y(:,2)/10,'r');
>> xlabel('Time (seconds)');
```

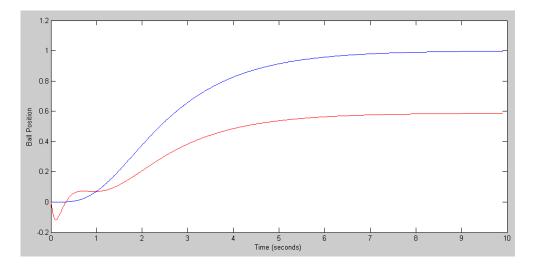


Step Resposne with respect to d: Position (blue) & U/10 (red) The system can reject a constant disturbance

- 4) Implement your control law on the nonlinear ball and beam system
 - With R = 1 and the mass of the ball being 0.5kg, and
 - With R = 1 and the mass of the ball being 0.6kg



Step response when m = 0.5kg. Position (blue) and U/10 (red)

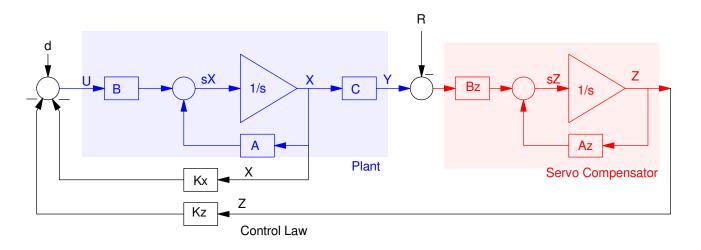


Step response when m = 0.6kg. Position (blue) and U/10 (red)

Code:

```
% Ball & Beam System
% Lecture #16
% Servo Compensators at DC
X = [0, 0, 0, 0]';
Z = 0;
dt = 0.01;
t = 0;
% Full-State Feedback Gains
              48.8540 -5.6397 20.0000];
Kx = [-7.2211]
Kr = -2.3211;
% Servo Compensator Gains
Kx = [ -84.6071 200.9500 -71.9893
                                      36.6750];
Kz = -28.7143;
n = 0;
y = [];
while (t < 9.9)
  Ref = 1;
   U = -Kz * Z - Kx * X;
   %U = Kr*Ref - Kx*X;
   dX = BeamDynamics(X, U);
   dZ = X(1) - Ref;
  X = X + dX * dt;
  Z = Z + dZ * dt;
   t = t + dt;
   y = [y; U/10, X(1)];
   n = mod(n+1, 5);
   if(n == 0)
     BeamDisplay(X, Ref);
      end
   end
t = [1:length(y)]' * dt;
plot(t,y(:,1),'r',t,y(:,2),'b');
xlabel('Time (seconds)');
ylabel('Ball Position');
```

Servo Compensators with Sinulsoidal Set-Points



0

0

0

0

0

- 5) Assume a 1 rad/sec disturbance and/or set point (R). Design a feedback control law that results in
 - The ability to track a constant set point (R = sin(t))
 - The ability to reject a constant disturbance (d = sin(t)), and ٠
 - A 2% settling time of 6 seconds

First, input the plant and servo compensator

>> A6 = [A, zeros(4, 2); Bz*C, Az]

```
>> A = [0, 0, 1, 0; 0, 0, 0, 1; 0, -7, 0, 0; -1.96, 0, 0, 0];
>> B = [0;0;0;0.4];
>> C = [1, 0, 0, 0];
>> Az = [0, 1; -1, 0];
>> Bz = [1;1];
```

Create the augmented system

1.0000 0 0 0 0 0 1.0000 0 0 0 0 -7.0000 0 0 0 -1.96000 0 0 0 1.0000 0 0 0 0 1.0000 0 0 1.0000 0 -1.0000>> B6 = [B; 0*Bz]0 0 0 0.4000 0 0 >> B6r = [0*B; -Bz] 0 0 0 0 -1 -1

Find the full-state feedback gains

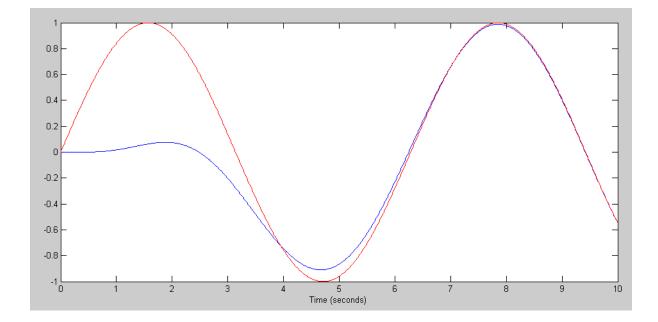
>> K6 = ppl(A6, B6, [-0.67+j, -0.67-j, -2, -3, -4, -5])
Kx
Kz
K6 = -125.9796 225.5223 -90.7445 38.3500 6.3052 -52.6787

6) For the linear system, plot the response

- With R(t) = sin(t), and
- With d(t) = sin(t)

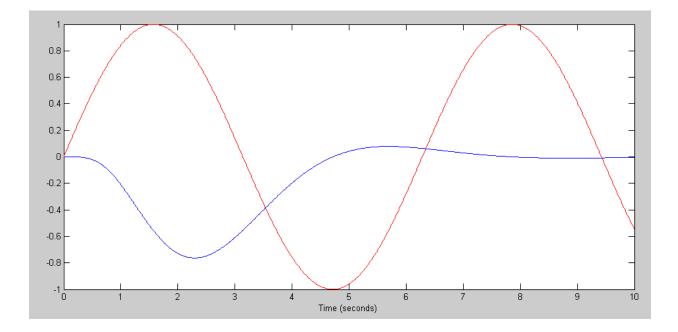
Take the response to a sinusoidal input using the step3() command (user created function in Matlab)

```
function [ y ] = step3( A, B, C, D, t, X0, U )
>> C6 = [C, 0,0 ; -K6];
>> D6 = [0;0];
>> X0 = zeros(6,1);
>> t = [0:0.01:10]';
>> R = sin(t);
>> y = step3(A6-B6*K6, B6r, C6, D6, t, X0, R);
>> plot(t,y(:,1),'b',t,R,'r');
>> xlabel('Time (seconds)');
```



Response to a sinusoidal set point. Output (blue) and set point (red)

```
>> y = step3(A6-B6*K6, B6, C6, D6, t, X0, R);
>> plot(t,y(:,1)*100,'b',t,R,'r');
>> xlabel('Time (seconds)');
```

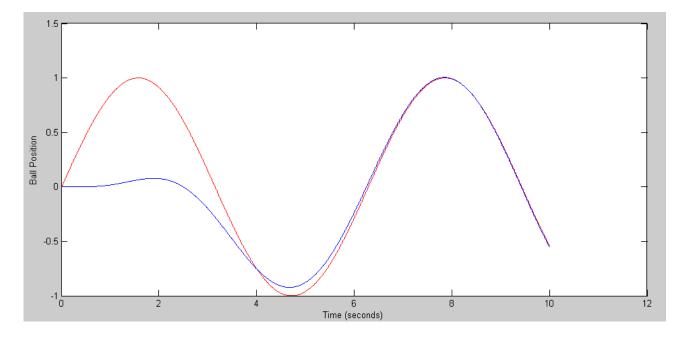


Response to a sinusoidal disturbance. Disturbance, D (red) and Position*100 (blue)

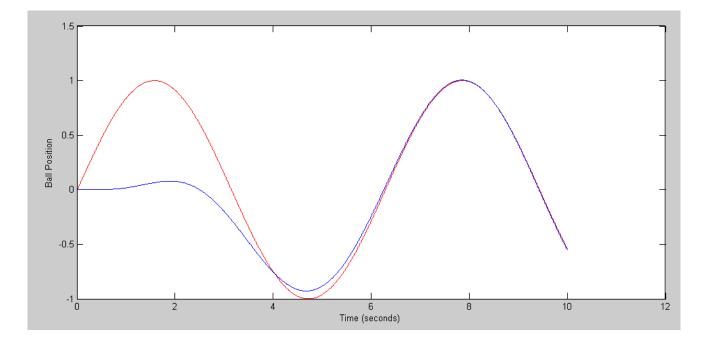
As expected, the sevo compensator

- Tracks a 1 rad/sec set point, and
- Rejects a 1 rad/sec disturbance.

- 7) Implement your control law on the nonlinear ball and beam system
 - With R = sin(t) and the mass of the ball being 0.5kg (nominal), and
 - With R = sin(t) and the mass of the ball being 0.6kg (ball has an extra 0.1kg)



Tracking a 1 rad/sec set point when m = 0.5kg



Tracking a 1 rad/sec set point when m = 0.6kg

Code:

```
% Ball & Beam System
% Lecture #17
% Servo Compensators at AC
X = [0, 0, 0, 0]';
Z = zeros(2,1);
dt = 0.01;
t = 0;
% Servo Compensator Gains
Kx = [-125.9796 \ 225.5223 \ -90.7445 \ 38.3500];
Kz = [6.3052 - 52.6787];
Az = [0, 1; -1, 0];
Bz = [1;1];
n = 0;
y = [];
while (t < 10)
   Ref = 1 \times \sin(t);
   U = -Kz * Z - Kx * X;
   dX = BeamDynamics(X, U);
   dZ = Az * Z + Bz * (X(1) - Ref);
   X = X + dX * dt;
   Z = Z + dZ * dt;
   t = t + dt;
   y = [y; Ref, X(1)];
   n = mod(n+1, 5);
   if(n == 0)
      BeamDisplay(X, Ref);
      end
   end
t = [1:length(y)]' * dt;
plot(t,y(:,1),'r',t,y(:,2),'b');
xlabel('Time (seconds)');
ylabel('Ball Position');
```