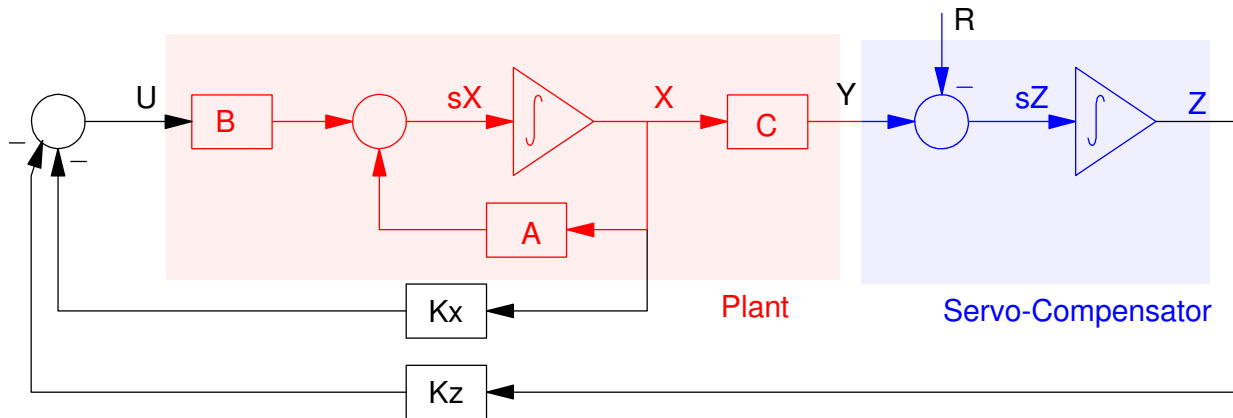


# ECE 463/663 - Homework #10

LQG Control with Servo Compensators. Due Monday, April 12th



**Cart and Pendulum (HW #7):** Use LQG methods to design a full-state feedback control law of the form

$$U = -K_z Z - K_x X$$

$$\dot{Z} = (x - R)$$

for the cart and pendulum system from homework #5 using LQG control so that

- You track constant setpoints,
- You reject constant disturbances,
- The 2% settling time is 6 seconds, and
- There is less than 5% overshoot for a step input.

1) Give the control law ( $K_x$  and  $K_z$ ) and explain how you chose  $Q$  and  $R$

The desired response has poles at (5% overshoot, damping ratio = 0.6901 or more)

$$s = -0.67 + j0.7$$

$$G_d = \left( \frac{0.939}{(s+0.67 \pm j0.7)} \right) = \left( \frac{0.939}{s^2 + 1.33s + 0.939} \right)$$

$Q$  and  $R$  are adjusted until the plant & feedback has about the same step response as  $G_d$

- Start with increasing the weighting on  $Z$  until you're fast enough ( $Q(5,5) = 50$ )
- Increase the weighting on  $Y$  to reduce the overshoot ( $Q(1,1) = 10$ )

## Matlab Code

```
A = [0,0,1,0;0,0,0,1;0,-39.2,0,0;0,49,0,0];
B = [0;0;1;-1];
C = [1,0,0,0];

A5 = [A, 0*B ; C, 0];
B5 = [B; 0];
B5r = [0*B; -1];
C5 = [C, 0];

Gd = tf(0.939,[1,1.33,0.939]);

K5 = lqr(A5, B5, diag([10,0,0,0,50]), 1);

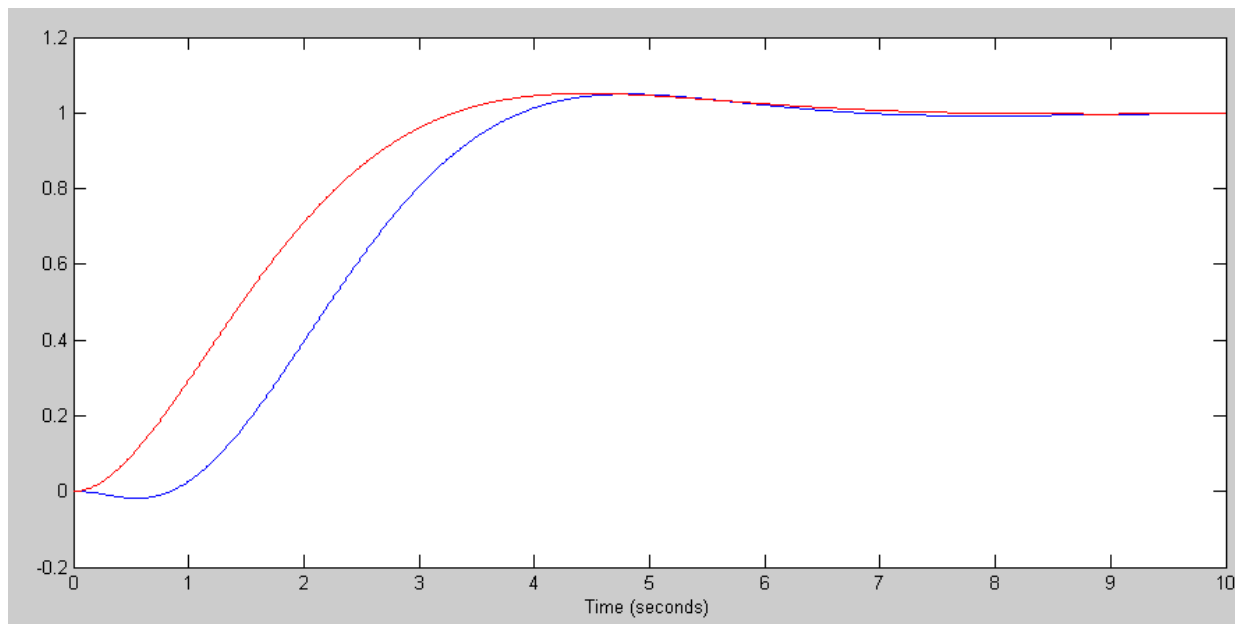
t = [0:0.01:10]';
G = ss(A5-B5*K5, B5r, C5, 0);
y = step(G,t);
yd = step(Gd, t);

plot(t,y,'b',t,yd,'r');
xlabel('Time (seconds)');
eig(A - B*Kx)

>> K5

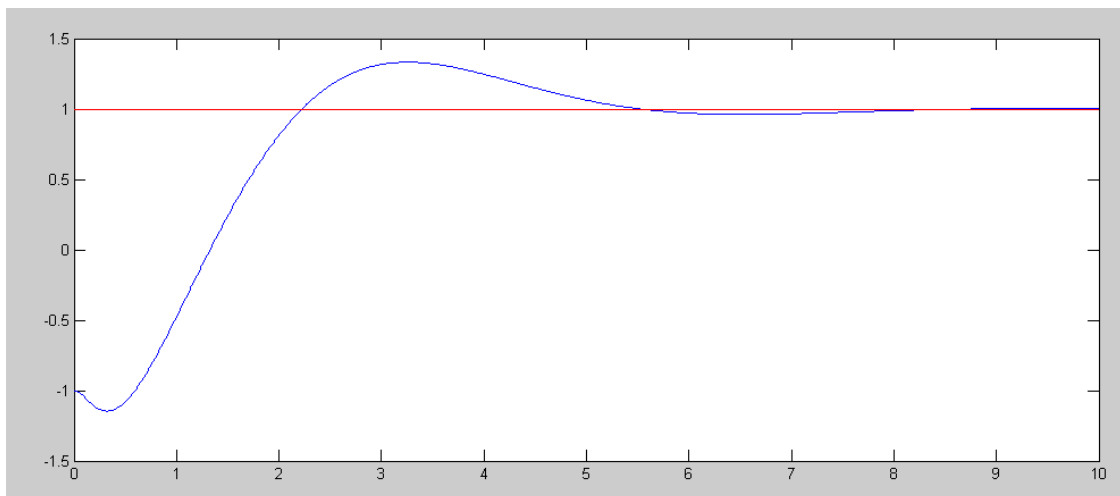
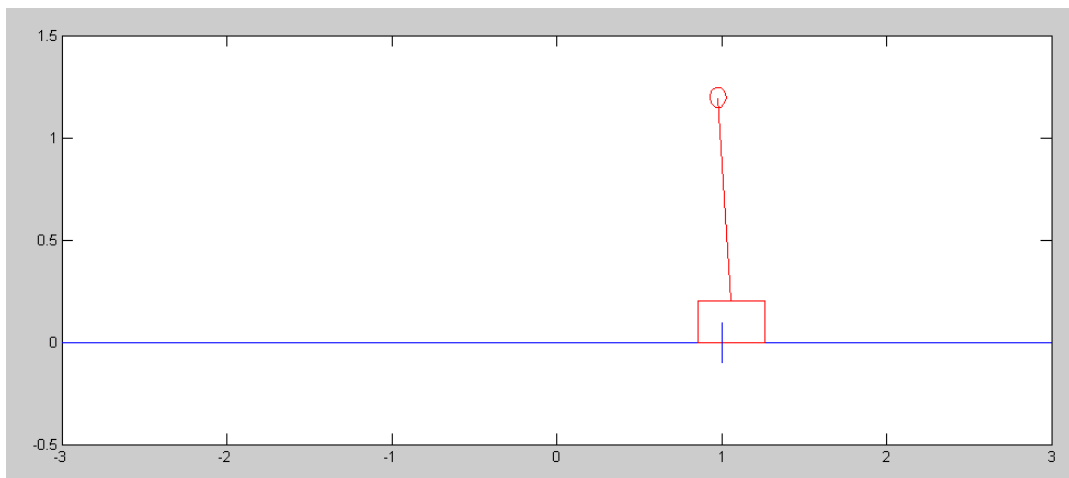
K5 = -15.5841 -149.1354 Kx -16.4659 -32.8092 Kz -7.0711
```

## 2) Plot the step response of the linear system



Desired Step Response (red) & Linear System's Step Response (blue)

3) Check your design with the nonlinear simulation of the cart and pendulum system.



**Ball and Beam (HW #7):** Use LQG methods to design a full-state feedback control law of the form

$$U = -K_z Z - K_x X$$

$$\dot{Z} = (x - R)$$

for the ball and beam system from homework #5 using LQG control so that

- You track constant setpoints,
- You reject constant disturbances,
- The 2% settling time is 6 seconds, and
- There is less than 5% overshoot for a step input.

4) Give the control law ( $K_x$  and  $K_z$ ) and explain how you chose Q and R

Same as problem #1 but using the (A,B,C,D) matrix for the ball and beam system

- Start with Q weighting Z (  $Q(5,5) = 1$  ). Adjust until it's fast enough.
- The resulting response is pretty close, so you're done

Code:

```
A = [0, 0, 1, 0; 0, 0, 0, 1; 0, -7, 0, 0; -1.96, 0, 0, 0];
B = [0; 0; 0; 0.4];
C = [1, 0, 0, 0];

A5 = [A, 0*B ; C, 0];
B5 = [B; 0];
B5r = [0*B; -1];
C5 = [C, 0];

Gd = tf(0.939, [1, 1.33, 0.939]);

K5 = lqr(A5, B5, diag([0, 300, 0, 0, 40]), 1);

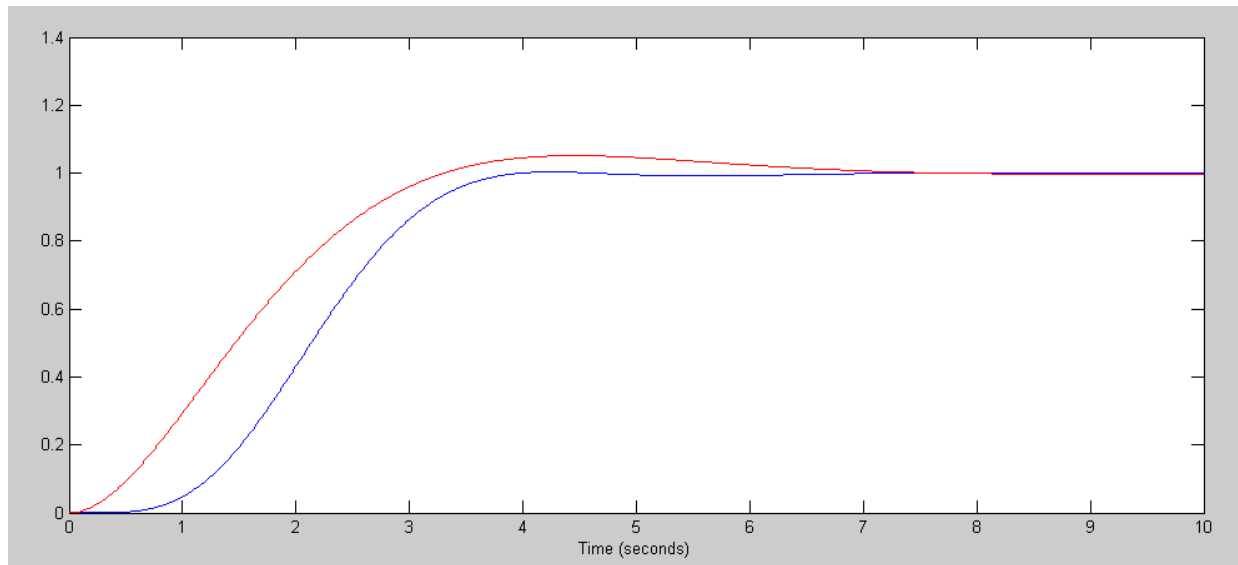
t = [0:0.01:10]';
G = ss(A5-B5*K5, B5r, C5, 0);
y = step(G, t);
yd = step(Gd, t);

plot(t, y, 'b', t, yd, 'r');
xlabel('Time (seconds)');
eig(A - B*Kx)

>> K5

K5 =    -18.6196    50.3087   -12.9825    15.8601    -6.3246
```

5) Plot the step response of the linear system



Desired Step Response (red) & Linear System's Step Response (blue)

6) Check your design with the nonlinear simulation of the cart and pendulum system.

