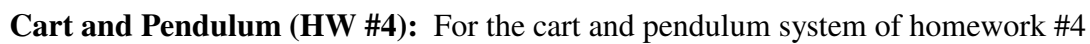


LQG Control with Servo Compensators. Due Monday, April 14th



Use LQG methods to design a full-state feedback control law of the form

- You track constant setpoints,
- You reject constant disturbances,
- The 2% settling time is 6 seconds, and
- There is less than 10% overshoot for a step input.

1) Give the control law (Kx and Kz) and explain how you chose Q and R

Define how the system *should* behave

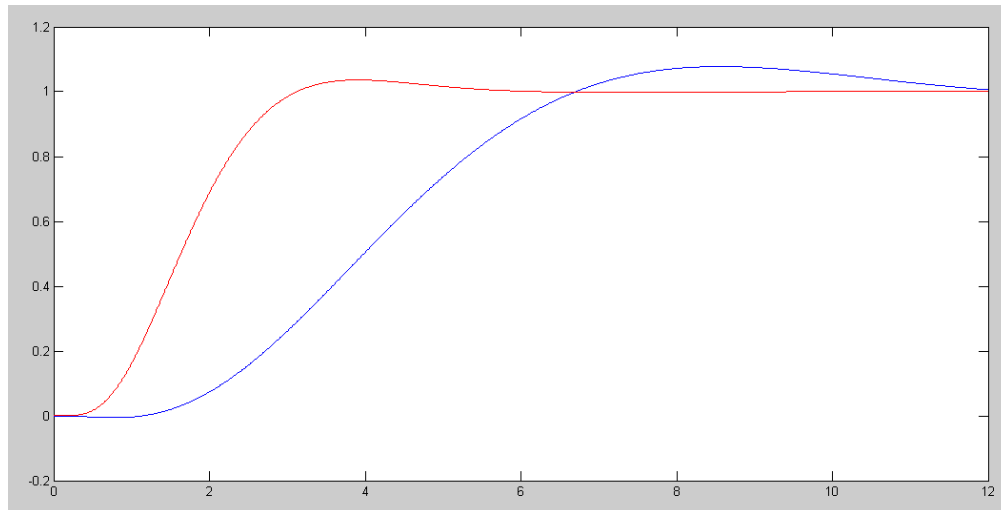
```
Gd = zpk([], [-1+j, -1-j, -4, -5, -6], 1);  
DC = evalfr(Gd, 0);  
Gd = Gd/DC;  
t = [0:0.01:12]';  
Yd = step(Gd, t);
```

Create an augmented system: plant plus a servo compensator with poles at $s=0$

```
A = [0, 0, 1, 0; 0, 0, 0, 1; 0, -19.6, 0, 0; 0, 19.6, 0, 0];  
B = [0; 0; 0.667; -0.444];  
C = [1, 0, 0, 0];  
  
A5 = [A, zeros(4, 1); C, 0];  
B5u = [B; 0];  
B5r = [0*B; -1];  
C5 = [1, 0, 0, 0, 0];  
D5 = 0;
```

Use LQR methods to find full-state feedback gains. Start with Q weighting the servo compensator alone:

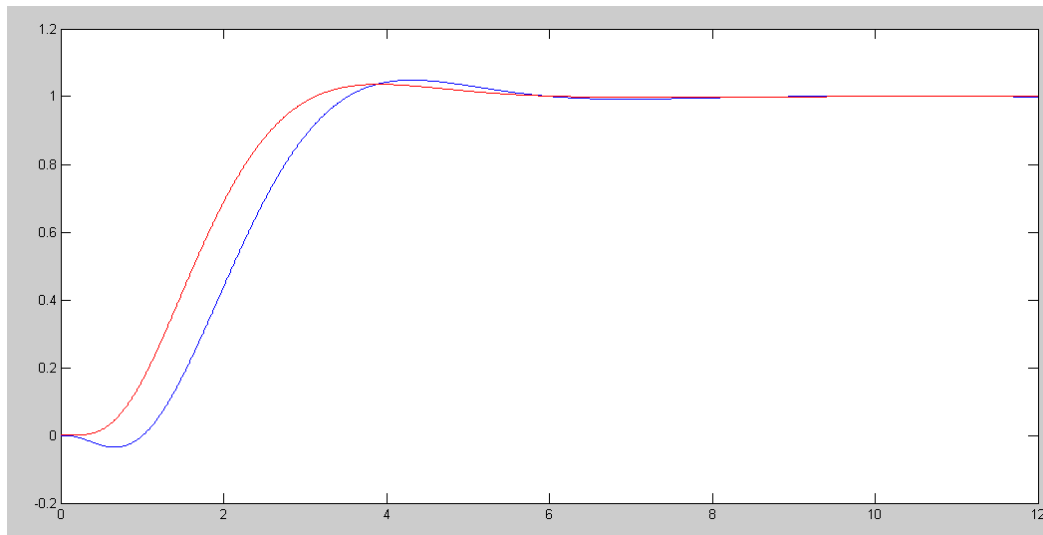
```
K5 = lqr(A5, B5u, diag([0, 0, 0, 0, 1]), 1);  
Gcl = ss(A5-B5u*K5, B5r, C5, D5);  
Y = step(Gcl, t);  
  
plot(t, Y, 'b', t, Yd, 'r');
```



Red = desired response (Y_d), Blue = actual response ($Y = x = \text{position}$)

Adjust the weights on Q to speed up the system (weight on z) and reduce the overshoot (weight on x) until the response looks OK

```
K5 = lqr(A5, B5u, diag([10,0,0,0,100]), 1);  
Gcl = ss(A5-B5u*K5, B5r, C5, D5);  
Y = step(Gcl, t);  
  
plot(t, Y, 'b', t, Yd, 'r');
```



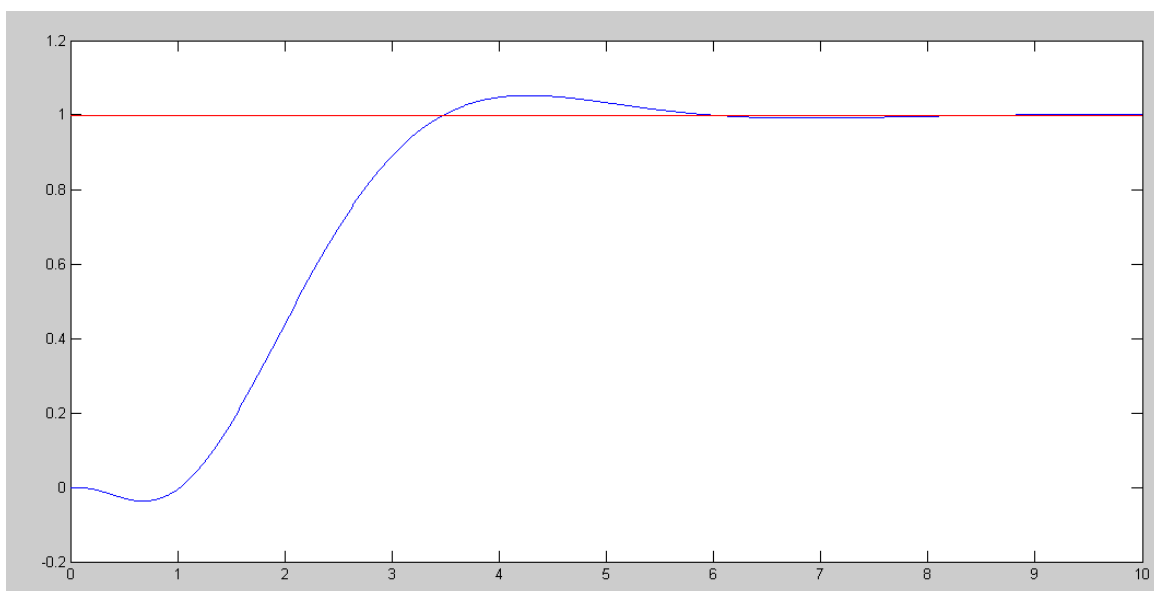
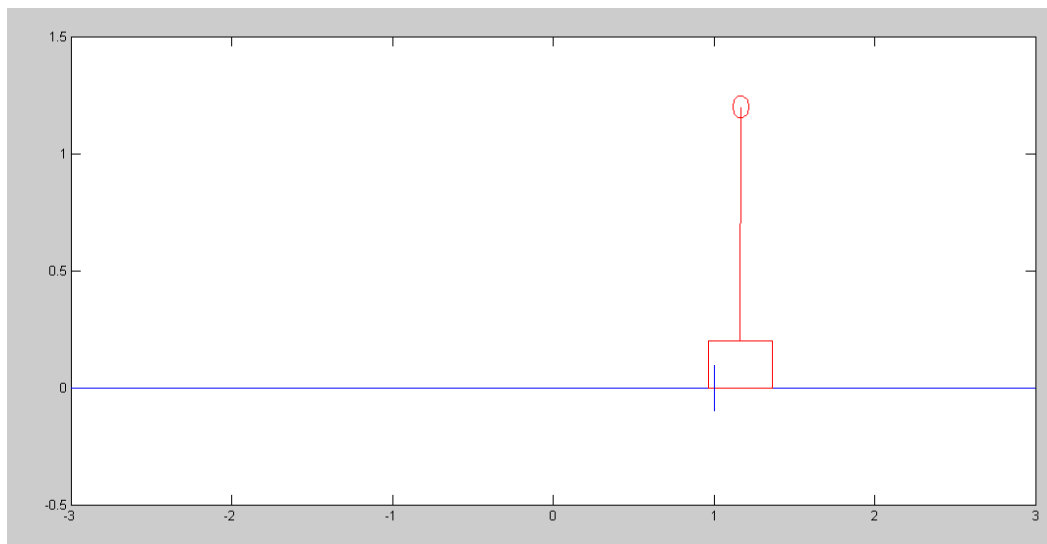
Desired response (red) & actual response (blue)

The resulting feedback gains and closed-loop pole locations

```
>> K5  
  
K5 = -21.1300 -182.9577 -21.8239 -58.8838 -10.0000  
  
>> eig(A5-B5u*K5)  
  
-4.4229 + 0.1110i  
-4.4229 - 0.1110i  
-0.7701 + 1.1246i  
-0.7701 - 1.1246i  
-1.2018  
  
>>
```

2) Plot the step response of the linear system
shown above

3) Check your design with the nonlinear simulation of the cart and pendulum system.



Ball and Beam (HW #4): For the ball and beam system of homework #4

$$s \begin{bmatrix} r \\ \theta \\ \dot{r} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -7 & 0 & 0 \\ -7.434 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} r \\ \theta \\ \dot{r} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0.345 \end{bmatrix} T$$

Use LQG methods to design a full-state feedback control law of the form

$$T = U = -K_z Z - K_x X$$

$$\dot{Z} = (x - R)$$

for the ball and beam system from homework #6 using LQG control so that

- You track constant setpoints,
- You reject constant disturbances,
- The 2% settling time is 6 seconds, and
- There is less than 10% overshoot for a step input.

4) Give the control law (Kx and Kx) and explain how you chose Q and R

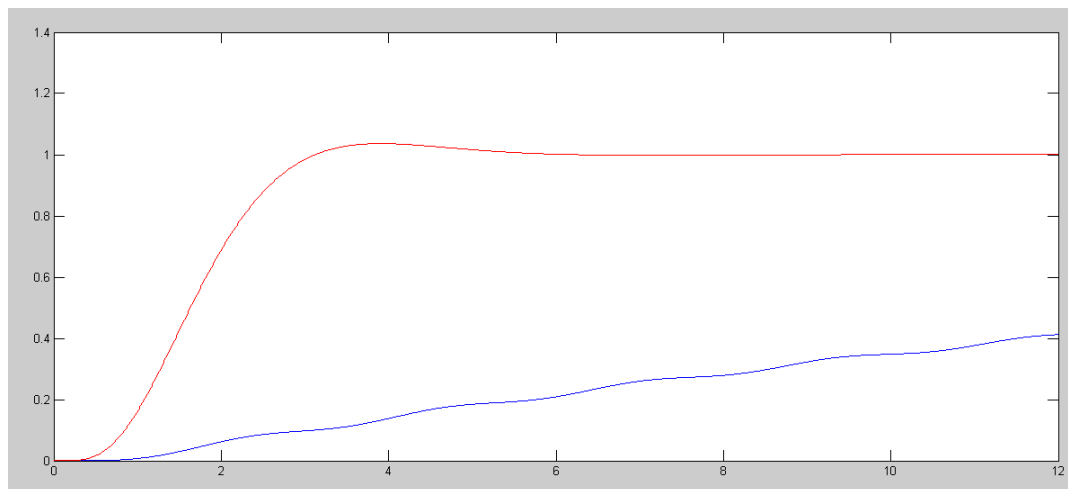
Same procedure as before.

- Start with adding a servo compensator with poles at $s = 0$
- Create the augmented system (plant & servo)
- Find full-state feedback gains using LQR methods
- Start with weighting the servo state

```
A = [0,0,1,0;0,0,0,1;0,-7,0,0;-7.424,0,0,0];  
B = [0;0;0;0.345];  
C = [1,0,0,0];
```

```
A5 = [A, zeros(4,1) ; C, 0];  
B5u = [B;0];  
B5r = [0*B;-1];  
C5 = [1,0,0,0,0];  
D5 = 0;
```

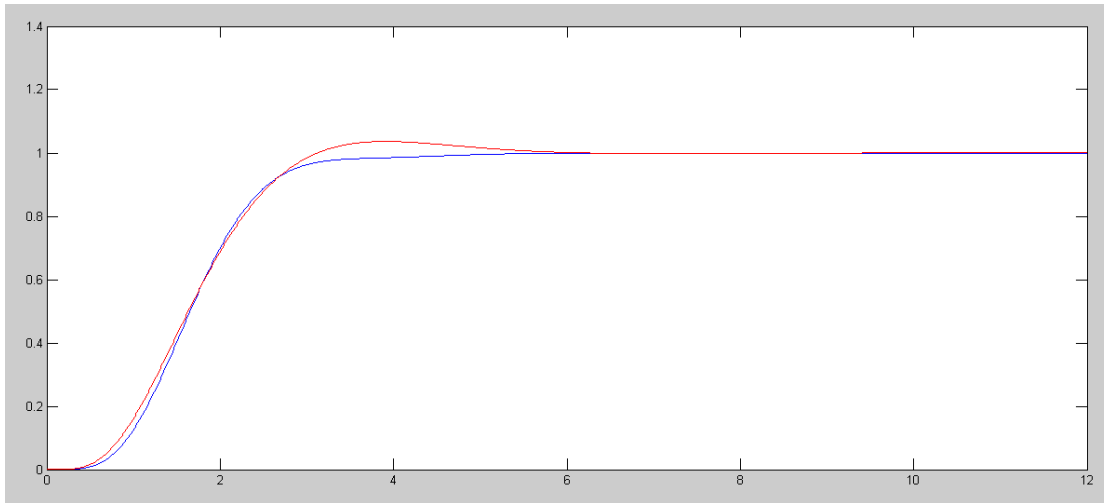
```
K5 = lqr(A5, B5u, diag([0,0,0,0,1]), 1);  
Gcl = ss(A5-B5u*K5, B5r, C5, D5);  
Y = step(Gcl, t);
```



Desired Response (red) & Actual Response

Increase the weightings in Q until the two are close

```
K5 = lqr(A5, B5u, diag([0,1600,0,0,900]), 1);  
Gcl = ss(A5-B5u*K5, B5r, C5, D5);  
Y = step(Gcl, t);
```



Red = Desired Step Response: Blue = Actual Step Response after tuning Q

The results are

```
>> K5
```

```
K5 =    -73.4849   108.1086   -37.2902    25.0343   -30.0000
```

```
>> eig(A5-B5u*K5)
```

```
-1.4516 + 2.2014i  
-1.4516 - 2.2014i  
-2.2031 + 1.7309i  
-2.2031 - 1.7309i  
-1.3274
```

5) Plot the step response of the linear system
shown above

6) Check your design with the nonlinear simulation of the cart and pendulum system.

